A COUPLED FINITE VOLUME SOLVER FOR THE SOLUTION OF LAMINAR/TURBULENT INCOMPRESSIBLE AND COMPRESSIBLE FLOWS

L. Mangani

Maschinentechnik CC Fluidmechanik und Hydromaschinen

Hochschule Luzern Technik& Architektur Technikumstrasse 21, CH-6048 Horw T +41 41 349 33 11, F +41 41 349 39 60 e-mail: luca.mangani@hslu.ch

> Lucerne University of Applied Sciences and Arts

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C. Bianchini

Dipartimento Energetica "S. Stecco"

Università degli Studi di Firenze Via Santa Marta 3, IT-50139 Firenze $T + 390554796575$ e-mail: cosimo.bianchini@htc.de.unifi.it

Presentation outline

- Background-State of the Art
- Coupled Algorithms vs. Segregated Approach
- Pressure Velocity Coupling
- Developments and present contribution
- Results and discussion
- Conclusions

Background-State of the Art

- The "engine" of computational fluid dynamics (CFD) is the pressurevelocity coupling algorithm that drives the fluid flow
	- Segregated approach: predictor-corrector loops
	- Coupled approach: pressure and velocity treated as a single vectorial unknown
- In past years efforts to develop more robust and efficient velocitypressure algorithms based on:
	- Choice of primitive variables density-based versus pressure-based
- For density but specially for pressure-based algorithms *the coupled versus segregated approach dichotomy has not been completely resolved yet!*
- Renewed interest in coupled solvers due to the increase in computers memory: commercial solvers

Coupled Algorithms vs. Segregated Approach

- Coupled Algorithm pros:
	- Fast Convergence: iterations only for momentum non linearity
	- Increase in efficiency for Steady State and time resolved arbitrary time step simulations
	- Less influence from the initial field, quasi initialization independence
	- Convergence speed grid independent
	- Fast convergence for simulations with extreme range of Mach:
		- \checkmark Ex: Stator Rotor Cavity applications

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Coupled Algorithms vs. Segregated Approach

- Coupled Algorithm cons:
	- Huge memory allocation
	- Less flexible
	- Critical choice of the linear solver for inner iterations
	- Critical choice of the controls parameter of the linear solver
	- Bad scalability for huge parallel calculations
	- Inefficient for time accurate simulations $Co < 1$

Pressure Velocity Coupling

- SIMPLE (Semi-Implicit Method for Pressure-Linked Equations)
	- Segregated approach for the pressure velocity coupling
- Coupled Pressure based U-p coupling without energy

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Pressure Velocity Coupling: Momentum Equation

- Momentum equation discretization
	- Pressure gradient becomes implicit
	- Pressure goes on the LHS

$$
\sum_{f=nb(P)}(\rho \textbf{v} \textbf{v} - \mu \nabla \textbf{v})_f \cdot \textbf{S}_f + \boxed{\sum_{f=nb(P)} p_f \textbf{S}_f} \neq \textbf{b}_P \Omega_P
$$

Gradient discretization

$$
p_{\rm f}=g_{\rm f}p_{\rm P}+(1-g_{\rm f})p_{\rm F}
$$

$$
\left(\begin{array}{c} a_{P}^{uu}u_{P}+\underline{a_{P}^{uv}v_{P}}+a_{P}^{up}p_{P}+\sum\limits_{F=NB(P)}a_{F}^{uu}u_{F}+\sum\limits_{F=NB(P)}a_{F}^{uv}v_{F}+\sum\limits_{F=NB(P)}a_{F}^{uv}v_{F}+\sum\limits_{F=NB(P)}a_{F}^{vv}p_{F}=b_{P}^{u}\end{array}\right)
$$

– Coupling coefficients for the momentum equations

$$
a_F^{uu}=a_F^{vv}=\mu_f\frac{\textbf{S}_f\cdot\textbf{S}_f}{\textbf{S}_f\cdot\textbf{d}_{PF}}+ \|\dot{m}_f,0\|\\ a_F^{uv}=\sum_{F=NB(P)}a_F^{vv}=\sum_{F=NB(P)}a_F^{vv}\\ \left.\begin{array}{cc} a_F^{up}=(1-g_f)S_f^x & a_F^{vp}=(1-g_f)S_f^y\\ a_P^{up}=\sum_{f=nb(P)}g_fS_f^x & a_P^{vp}=\sum_{f=nb(P)}g_fS_f^y\end{array}\right.
$$

Coupled algorithm: Comments

- If pressure equation is **NOT** introduced and the momentum and continuity equations are used:
	- We have a Saddle Block Matrix problem

$$
\frac{\partial \mathbf{u}}{\partial t} + \nabla \bullet (\mathbf{u}\mathbf{u}) - \nabla \bullet (\nu \nabla \mathbf{u}) = -\nabla p
$$

 $\nabla_{\bullet} \mathbf{u} = 0$

$$
\begin{bmatrix} [A_{\mathbf{u}}] & [\nabla(.)] \\ [\nabla_{\bullet}(.)] & \boxed{0} \end{bmatrix} \begin{bmatrix} \mathbf{u} \\ p \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}
$$

- Since no pressure equation is derived, zeros are present in the main diagonal of the discretized continuity equation
- Leading to an ill conditioned system of equations

Pressure Velocity Coupling: Continuity Equation

- The use of the pressure equation solve the Saddle block matrix issue
	- Pressure equation derived from the continuity and momentum equation

 $\sum \ \rho_f \mathbf{v}_f \cdot \mathbf{S}_f = 0$ $f = nb(P)$

- Using Rhie-Chow interpolation
- Discretized equation

$$
\sum_{f=nb(P)} \rho_f [\overline{\boldsymbol{v}_f} - \overline{\boldsymbol{D}_f}(\nabla p_f - \overline{\nabla p_f})] \cdot \boldsymbol{S}_f = 0
$$

 $a_{\rm P}^{pu} = \sum g_{\rm f} S_{\rm f}^{x} a_{\rm P}^{pv} = \sum g_{\rm f} S_{\rm f}^{y}$

$$
\sum_{f=nb(P)} \rho_f(-\overline{\bm{D}_f}\nabla p_f)\cdot \bm{S}_f + \sum_{f=nb(P)} \rho_f \overline{\bm{v}_f}\cdot \bm{S}_f = \sum_{f=nb(P)} \rho_f(-\overline{\bm{D}_f\nabla p_f})\cdot \bm{S}_f
$$

– Pressure-velocity coupling coefficients derived from mass fluxes imbalance $a_F^{pu} = (1 - g_f)S_f^x$ $a_F^{pv} = (1 - g_f)S_f^y$

$$
a_{P}^{pp}p_{P}+a_{P}^{pu}u_{P}+a_{P}^{pv}v_{P}+\sum_{F=NB(P)}a_{F}^{pp}p_{F}+\sum_{F=NB(P)}a_{F}^{pu}u_{F}+\sum_{F=NB(P)}a_{F}^{pv}v_{F}=b_{P}^{p}
$$

Fifth OpenFOAM Workshop, June 22-24 2010, Gothenburg, Sweden

 $f = nb(P)$

 $f = nb(P)$

Matrix Form

- Cell based variable storage
	- Global matrix is composed of a 4x4 local matrix for each cell (equivalent to fvMatrix.A() coeffs)
	- Matrix pattern expanded to include the influence of neighboring control volumes (equivalent to fvMatrix.H() coeffs)
- Local matrix structure:

UU **UP** VV VP $\begin{bmatrix} VV & VP \\ WW & WP \\ W & PV & PP \end{bmatrix} = \begin{bmatrix} S_U \\ S_V \\ S_W \\ S_W \end{bmatrix}$ PU

OpenFOAM Development

- OpenFOAM current version cannot handle block matrix
	- lduMatrix addressing is referred to the mesh size
- Development of a generic matrix interface to handle external linear solvers

Compressibility and Turbulence

If flow is in compressible regime, the change in fluid density should be taken into account

$$
\rho=\rho^*+\rho'=\rho^*+\frac{\partial\rho}{\partial p}\,p'
$$

- The convection flux should also be modified in the pressure equation
- Turbulence model was added based on k-ω SST model with Low-Reynolds or automatic wall treatment
	- $-$ k and ω are solved also in a coupled way
	- A block sparse matrix for turbulence is solved too

Results

- Comparison is performed between coupled and segregated solver on reference test cases $(coupled = solid line, separated = dash line)$
- Convergence speed is checked plotting normalized residuals $\sqrt{2}$

$$
R_{\phi} = \frac{\sqrt{\sum} (\phi_{old} - \phi_{new})^2}{\sqrt{\sum} (\phi_{old} - \widetilde{\phi}_{old})^2}
$$

- Uniform initialization, energy and turbulence activated from beginning
- Incompressible and compressible formulation
- Inviscid, laminar and turbulent test cases
- Periodic boundaries

Turbulent Flat Plate

- Adiabatic 2D flat plate at $Ma_{in}=0.2$
- Turbulent boundary layer integrated up to the wall $y^+ \approx 0.1$
- Fixed $localCo = 5000$

k/k $_{\rm max}$ [-]

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 $1.0 -$

 $0.8 -$

Backward Facing Step Test

- Incompressible isothermal turbulent flow
- Automatic wall treatment
- No time derivative $(Co = \infty)$
- Explicit relaxation factors

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GAMM Test

- Inviscid transonic test case
- Confined circular bump at $Ma_{\infty}=0.675$
- Fixed localCo = 600

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NACA 0012

- Isolated 2D profile
- Inviscid test case
- Transonic test Ma_∞=0.75, $a = 4^{\circ}$
- Circular domain, inletOutlet BC
- Tetrahedral mesh
- Fixed localCo $= 600$
- Results in terms of dimensionless pressure

$$
C_p = \frac{p - p_{\infty}}{\frac{1}{2} \rho_{\infty} U_{\infty}^2}
$$

NACA 0012

- Only 300 iterations to reach the convergence
- Even though more diffused the shock location is well predicted
- Profile load correctly reproduced

Goldman Test

- 2D linear cascade -> fully implicit coupled boundary
- Highly compressible turbulent test case $Ma_{in}=0.2$
- average $y^+=50$ automatic wall treatment
- Adiabatic surface, fixed velocity and static pressure
- Fixed $localCo = 500$

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Goldman Test

- Pressure profile coincident with segregated
- Good agreement with experimental values
- Drift respect to other codes due to BC and turbulence model
- Faster convergence than segregated
- Lower level of residuals than segregated

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Conclusions

- A turbulent compressible and incompressible coupled solver has been developed in OF framework
	- Consistent results compared to the segregated solver were reproduced among:
		- \checkmark Inviscid/viscous/turbulent, compressible/incompressible, LowReynolds/WallFunction, periodic flows
- Improved convergence and stability respect to segregated solvers
- Main Drawbacks
	- Speed of linear solver can be improved
	- Great amount of memory allocation can be reduced with a more efficient implementation
	- Further generalization in the code to be achieved
- Applications to more complex cases to be achieved
	- Multi-phase, Combustion, FSI,…

