A COUPLED FINITE VOLUME SOLVER FOR THE SOLUTION OF LAMINAR/TURBULENT INCOMPRESSIBLE AND COMPRESSIBLE FLOWS

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Presentation outline

- Background-State of the Art
- Coupled Algorithms vs. Segregated Approach
- Pressure Velocity Coupling
- Developments and present contribution
- Results and discussion
- Conclusions







Background-State of the Art

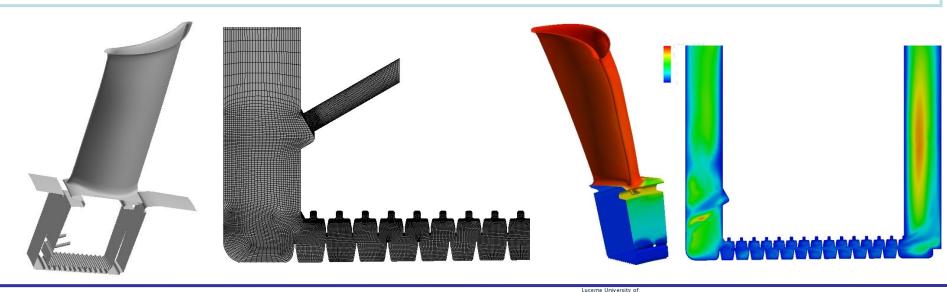
- The "engine" of computational fluid dynamics (CFD) is the pressurevelocity coupling algorithm that drives the fluid flow
 - Segregated approach: predictor-corrector loops
 - Coupled approach: pressure and velocity treated as a single vectorial unknown
- In past years efforts to develop more robust and efficient velocitypressure algorithms based on:
 - Choice of primitive variables density-based versus pressure-based
- For density but specially for pressure-based algorithms <u>the coupled</u> <u>versus segregated approach dichotomy has not been completely</u> <u>resolved yet!</u>
- Renewed interest in coupled solvers due to the increase in computers memory: commercial solvers





Coupled Algorithms vs. Segregated Approach

- Coupled Algorithm pros:
 - Fast Convergence: iterations only for momentum non linearity
 - Increase in efficiency for Steady State and time resolved arbitrary time step simulations
 - Less influence from the initial field, quasi initialization independence
 - Convergence speed grid independent
 - Fast convergence for simulations with extreme range of Mach:
 - ✓ Ex: Stator Rotor Cavity applications

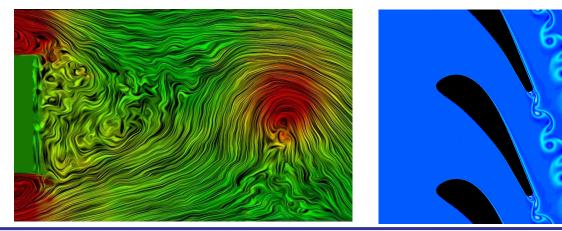


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Coupled Algorithms vs. Segregated Approach

- Coupled Algorithm cons:
 - Huge memory allocation
 - Less flexible
 - Critical choice of the linear solver for inner iterations
 - Critical choice of the controls parameter of the linear solver
 - Bad scalability for huge parallel calculations
 - Inefficient for time accurate simulations Co < 1

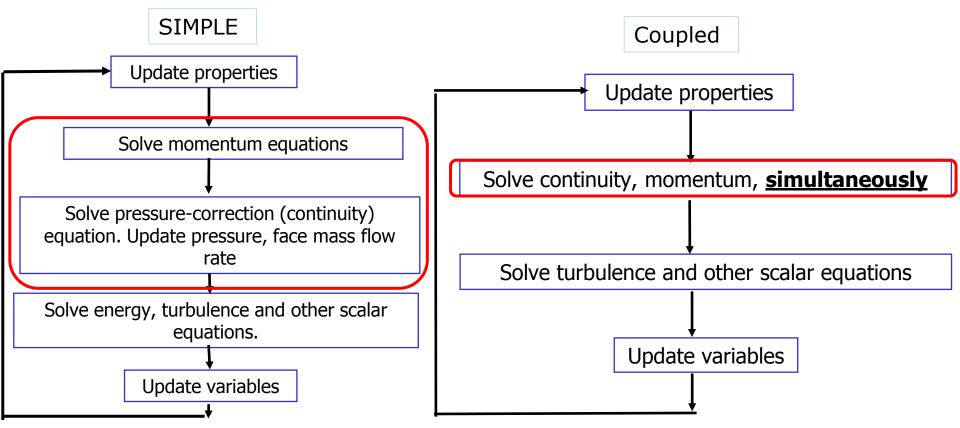






Pressure Velocity Coupling

- SIMPLE (Semi-Implicit Method for Pressure-Linked Equations)
 - Segregated approach for the pressure velocity coupling
- Coupled Pressure based U-p coupling without energy



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Pressure Velocity Coupling: Momentum Equation

- Momentum equation discretization
 - Pressure gradient becomes implicit
 - Pressure goes on the LHS

$$\sum_{\mathbf{F}=\mathbf{nb}(\mathbf{P})} (\rho \mathbf{v} \mathbf{v} - \mu \nabla \mathbf{v})_{\mathbf{f}} \cdot \mathbf{S}_{\mathbf{f}} + \sum_{\mathbf{f}=\mathbf{nb}(\mathbf{P})} p_{\mathbf{f}} \mathbf{S}_{\mathbf{f}} = \mathbf{b}_{\mathbf{P}} \Omega_{\mathbf{P}}$$

Gradient discretization

$$p_{\mathrm{f}} = g_{\mathrm{f}} p_{\mathrm{P}} + (1 - g_{\mathrm{f}}) p_{\mathrm{F}}$$

$$\begin{pmatrix}
a_{P}^{uu}u_{P} + \underline{a_{P}^{uv}}v_{P} + \underline{a_{P}^{up}}p_{P} + \sum_{F=NB(P)}a_{F}^{uu}u_{F} + \sum_{\underline{F=NB(P)}}a_{F}^{uv}v_{F} + \sum_{\underline{F=NB(P)}}a_{F}^{uv}v_{F} + \sum_{\underline{F=NB(P)}}a_{F}^{vv}v_{F} +$$

Coupling coefficients for the momentum equations

$$\begin{aligned} a_{F}^{uu} &= a_{F}^{vv} = \mu_{f} \frac{\mathbf{S}_{f} \cdot \mathbf{S}_{f}}{\mathbf{S}_{f} \cdot \mathbf{d}_{PF}} + \|\dot{m}_{f}, \mathbf{0}\| \\ a_{P}^{uu} &= \sum_{F=NB(P)} a_{F}^{uu} \quad a_{P}^{vv} = \sum_{F=NB(P)} a_{F}^{vv} \end{aligned} \qquad \begin{aligned} a_{F}^{up} &= (1 - g_{f})S_{f}^{x} \quad a_{F}^{vp} = (1 - g_{f})S_{f}^{y} \\ a_{P}^{up} &= \sum_{f=nb(P)} g_{f}S_{f}^{x} \quad a_{P}^{vp} = \sum_{f=nb(P)} g_{f}S_{f}^{y} \end{aligned}$$





Coupled algorithm: Comments

- If pressure equation is **NOT** introduced and the momentum and continuity equations are used:
 - We have a Saddle Block Matrix problem

$$\frac{\partial \mathbf{u}}{\partial t} + \nabla \mathbf{\bullet} (\mathbf{u} \mathbf{u}) - \nabla \mathbf{\bullet} (\nu \nabla \mathbf{u}) = -\nabla p$$

 $\nabla \bullet \mathbf{u} = 0$

$$\begin{bmatrix} [A_{\mathbf{u}}] & [\nabla(.)] \\ [\nabla \bullet(.)] & [0] \end{bmatrix} \begin{bmatrix} \mathbf{u} \\ p \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

- Since no pressure equation is derived, zeros are present in the main diagonal of the discretized continuity equation
- Leading to an ill conditioned system of equations







Pressure Velocity Coupling: Continuity Equation

- The use of the pressure equation solve the Saddle block matrix issue
 - Pressure equation derived from the continuity and momentum equation

 $\sum_{f=nb(P)} \rho_f \boldsymbol{v}_f \cdot \boldsymbol{S}_f = \boldsymbol{0}$

- Using Rhie-Chow interpolation
- Discretized equation

$$\sum_{\mathbf{f}=\mathbf{n}\mathbf{b}(\mathbf{P})} \rho_{\mathbf{f}}[\overline{\mathbf{v}_{\mathbf{f}}} - \overline{\mathbf{D}_{\mathbf{f}}}(\nabla p_{\mathbf{f}} - \overline{\nabla p_{\mathbf{f}}})] \cdot \mathbf{S}_{\mathbf{f}} = \mathbf{0}$$

 $a_{\rm P}^{pu} = \sum g_{\rm f} S_{\rm f}^{x} \quad a_{\rm P}^{pv} = \sum g_{\rm f} S_{\rm f}^{y}$

$$\sum_{f=nb(P)} \rho_f(-\overline{\mathbf{D}_f} \nabla p_f) \cdot \mathbf{S}_f + \sum_{f=nb(P)} \rho_f \overline{\mathbf{v}_f} \cdot \mathbf{S}_f = \sum_{f=nb(P)} \rho_f(-\overline{\mathbf{D}_f} \nabla p_f) \cdot \mathbf{S}_f$$

- Pressure-velocity coupling coefficients derived from mass fluxes imbalance $a_{\rm F}^{pu} = (1 - g_{\rm f})S_{\rm f}^x$ $a_{\rm F}^{pv} = (1 - g_{\rm f})S_{\rm f}^y$

$$a_{P}^{pp}p_{P} + a_{P}^{pu}u_{P} + a_{P}^{pv}v_{P} + \sum_{F=NB(P)} a_{F}^{pp}p_{F} + \sum_{F=NB(P)} a_{F}^{pu}u_{F} + \sum_{F=NB(P)} a_{F}^{pv}v_{F} = b_{P}^{p}$$





Matrix Form

- Cell based variable storage
 - Global matrix is composed of a 4x4 local matrix for each cell (equivalent to fvMatrix.A() coeffs)
 - Matrix pattern expanded to include the influence of neighboring control volumes (equivalent to fvMatrix.H() coeffs)
- Local matrix structure:

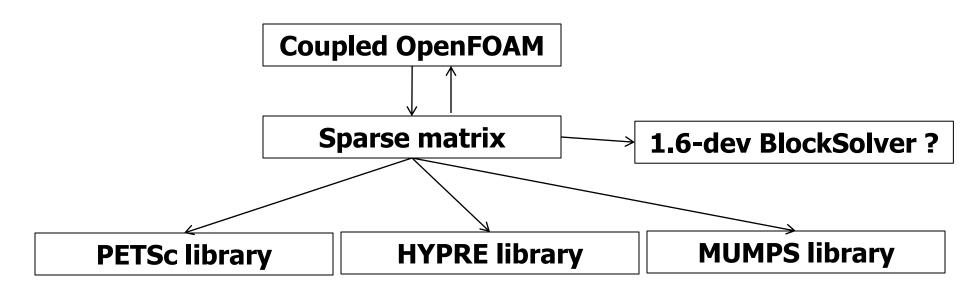
 $\begin{pmatrix} 1 & UU & UP \\ VV & VP \\ WW & WP \\ PU & PV & PW & PP \end{pmatrix} \begin{pmatrix} U \\ V \\ W \\ P \\ P \\ - \end{pmatrix} = \begin{pmatrix} S_{U} \\ S_{V} \\ S_{W} \\ S_{P} \\ - \end{pmatrix}$





OpenFOAM Development

- OpenFOAM current version cannot handle block matrix
 - IduMatrix addressing is referred to the mesh size
- Development of a generic matrix interface to handle external linear solvers







Compressibility and Turbulence

• If flow is in compressible regime, the change in fluid density should be taken into account

$$\rho = \rho^* + \rho' = \rho^* + \frac{\partial \rho}{\partial p} p'$$

- The convection flux should also be modified in the pressure equation
- Turbulence model was added based on k-ω SST model with Low-Reynolds or automatic wall treatment
 - k and ω are solved also in a coupled way
 - A block sparse matrix for turbulence is solved too







Results

- Comparison is performed between coupled and segregated solver on reference test cases
 (coupled = solid line, segregated = dash line)
- Convergence speed is checked plotting normalized residuals $\sqrt{\sum (\phi_{11} \phi_{22})^2}$

$$R_{\phi} = \frac{\sqrt{\sum (\phi_{old} - \phi_{new})^2}}{\sqrt{\sum (\phi_{old} - \tilde{\phi}_{old})^2}}$$

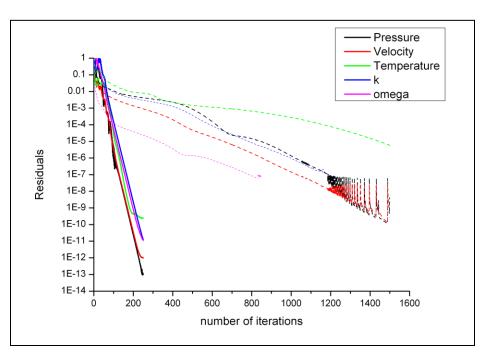
- Uniform initialization, energy and turbulence activated from beginning
- Incompressible and compressible formulation
- Inviscid, laminar and turbulent test cases
- Periodic boundaries

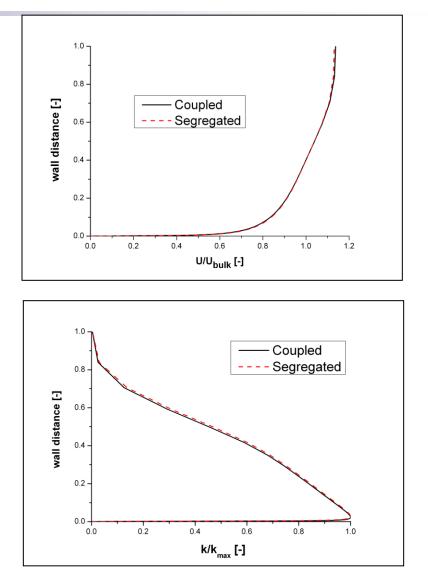




Turbulent Flat Plate

- Adiabatic 2D flat plate at Ma_{in}=0.2
- Turbulent boundary layer integrated up to the wall y⁺ ≈0.1
- Fixed localCo = 5000





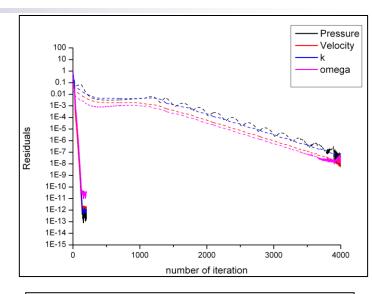
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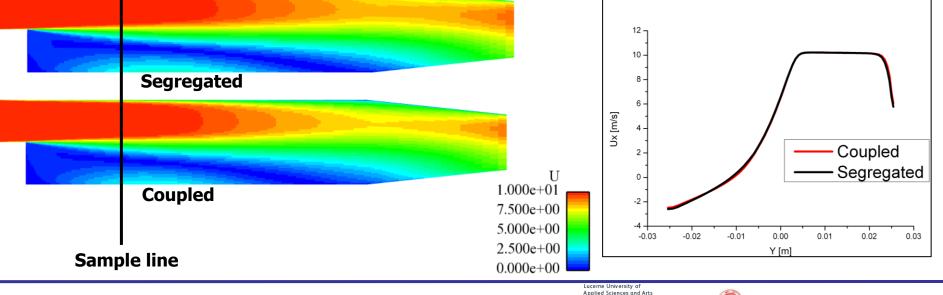
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Backward Facing Step Test

- Incompressible isothermal turbulent flow
- Automatic wall treatment
- No time derivative (Co=∞)
- Explicit relaxation factors

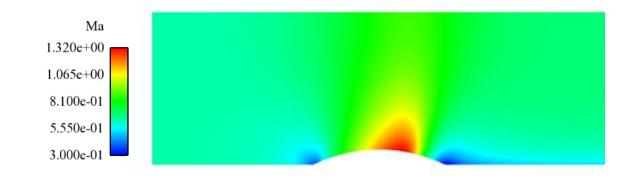




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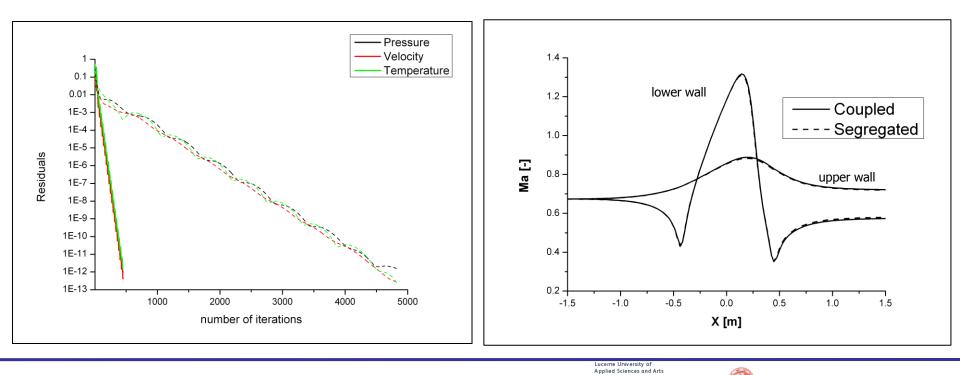
GAMM Test

- Inviscid transonic test case
- Confined circular bump at $Ma_{\infty}=0.675$
- Fixed localCo = 600



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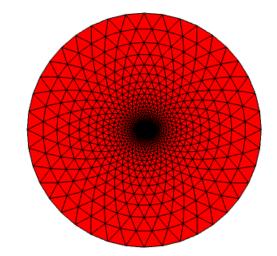


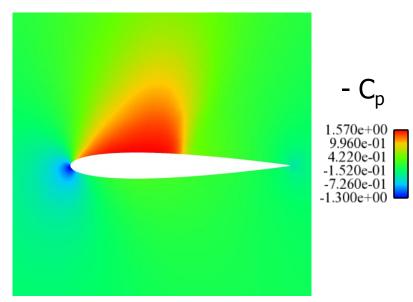


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- Isolated 2D profile
- Inviscid test case
- Transonic test $Ma_{\infty}=0.75, a = 4^{\circ}$
- Circular domain, inletOutlet BC
- Tetrahedral mesh
- Fixed localCo = 600
- Results in terms of dimensionless pressure

$$C_{p} = \frac{p p_{\infty}}{\frac{1}{2}\rho_{\infty}U_{\infty}^{2}}$$



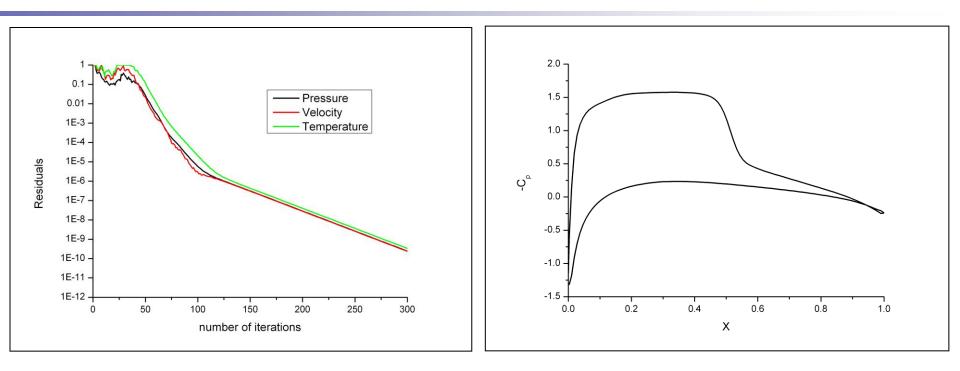


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- Only 300 iterations to reach the convergence
- Even though more diffused the shock location is well predicted
- Profile load correctly reproduced

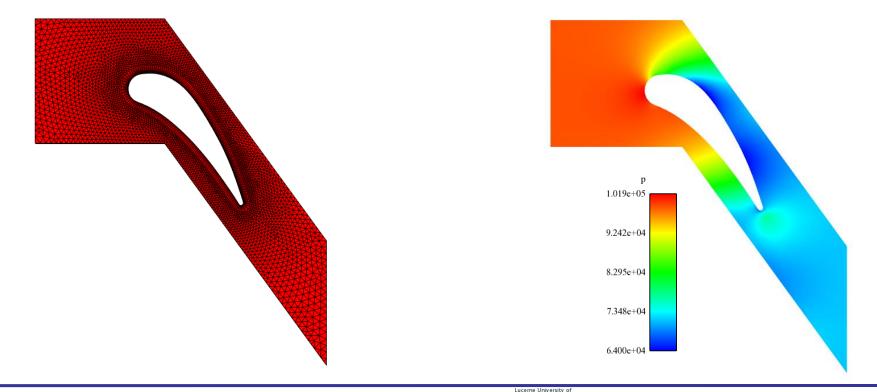






Goldman Test

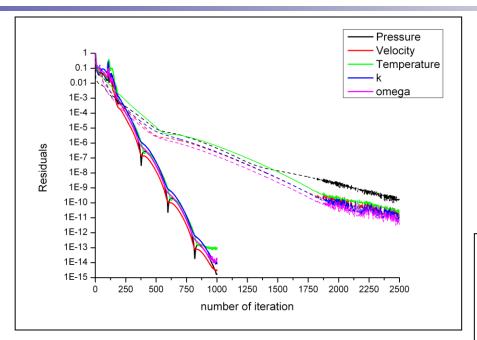
- 2D linear cascade -> fully implicit coupled boundary
- Highly compressible turbulent test case Ma_{in}=0.2
- average $y^+ = 50$ automatic wall treatment
- Adiabatic surface, fixed velocity and static pressure
- Fixed localCo = 500



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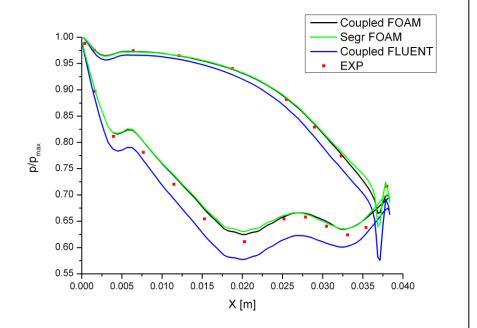
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Goldman Test



- Pressure profile coincident with segregated
- Good agreement with experimental values
- Drift respect to other codes due to BC and turbulence model

- Faster convergence than segregated
- Lower level of residuals than segregated



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Conclusions

- A turbulent compressible and incompressible coupled solver has been developed in OF framework
 - Consistent results compared to the segregated solver were reproduced among:
 - ✓ Inviscid/viscous/turbulent, compressible/incompressible, LowReynolds/WallFunction, periodic flows
- Improved convergence and stability respect to segregated solvers
- Main Drawbacks
 - Speed of linear solver can be improved
 - Great amount of memory allocation can be reduced with a more efficient implementation
 - Further generalization in the code to be achieved
- Applications to more complex cases to be achieved
 - Multi-phase, Combustion, FSI,...



