Simulating the combustion of gaseous fuels 6th OpenFoam Workshop Training Session

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This presentation shows how to use OpenFoam to simulate gas phase combustion



Overview

Theory

Tutorial case

Solution strategies

Validation





The focus of combustion simulation depends on the application





Burner design

Pollutant formation

Furnace operation/retrofit





The focus of the present tutorial is simulating a model flame

- Model flames are a basis to test and evaluate combustion solvers
- Tutorial case is a turbulent methane/air flame ("Flame D" from Sandia/TNF workshop)
- Solver applications used are rhoReactingFoam (PaSR model) edcSimpleFoam (EDC model)
- Validation with experimental data to assess the solver/model accuracy







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Combustion simulation is characterized by chemical reactions

Global reactions summarize the combustion process:

$$CH_4 + 2O_2 \Leftrightarrow CO_2 + 2H_2O$$

Detailed chemical mechanisms describe events on molecular level:

$$CH_4+O \Leftrightarrow OH+CH_3$$

 $CH_4 \Leftrightarrow CH_3+H$
 $CH_4+H \Leftrightarrow CH_3+H_2$
 $CH_4+OH \Leftrightarrow CH_3+H_2O$
:

from GRI-Mech 3.0 (325 reactions, 53 species)

- Chemical mechanisms need to be used within their specification limits, eg. GRI-Mech 3.0: methane/natural gas, T in 1000-2500 K, Φ in 0.1-5
- Detailed mechanism are more accurate (e.g. NO_x , ignition delay), but computationally much more expensive
 - → level of detail needs to be chosen by the user





Chemical reactions can be described with equilibrium or kinetic rates (incl. "infinite rate")

Equilibrium calculation depends only on thermodynamic data: h°, s°, cp° But concerning combustion, many things are not in equilibrium!

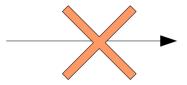
Chemical kinetics determine the reaction rate e.g. with an Arrhenius type formulation:

$$R = A T^{b} exp \left(-\frac{E}{\Re T} \right) C_{CH4} C_{O2}^{0.5}$$

"Infinite rate" chemistry is a special case, where reaction rates are assumed to be infitely fast







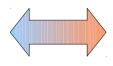






In turbulent flows, turbulence/chemistry interaction defines the reacting flow

Turbulent flow



Chemical kinetics

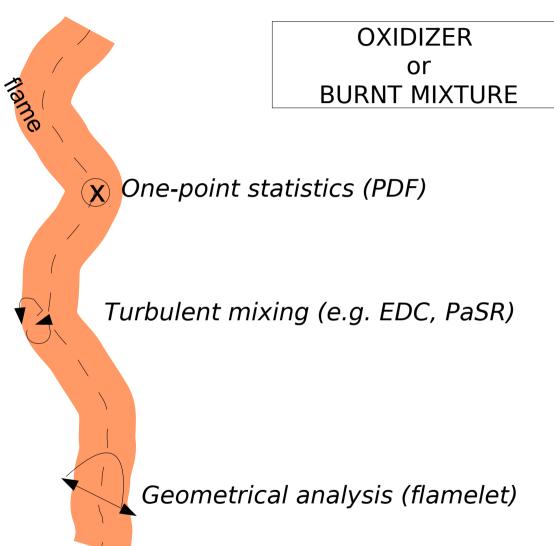
- Turbulence enhances mixing of species such as fuel, oxidizer and products
- Strong turbulence can suppress combustion
 - → local extinction

- In a laminar flow, combustion is controlled exclusively by chemical kinetics
- Combustion leads to flow acceleration→ modification of flow field



Different approaches exist to model the turbulence/chemistry interaction

FUEL or FRESH MIXTURE



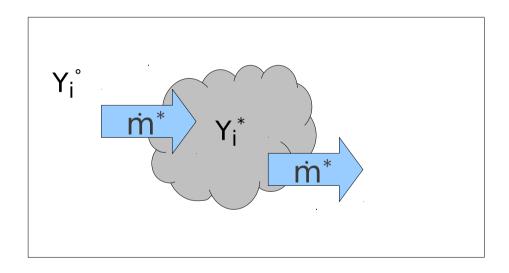
cf. Poinsot, Veynante "Theoretical and Numerical Combustion"





The Eddy-Dissipation Concept (EDC) assumes reactions in *fine* structures

$$\overline{R}_{i} = \overline{\rho} \frac{\gamma^{*} \dot{m}^{*}}{(1 - \gamma^{*})} (\overline{Y}_{i} - Y_{i}^{*})$$



Fraction of the flow occupied by *fine structures:*

Relation of mean, fine structure and surrounding state:

$$\overline{Y}_{i} = \gamma^{*} Y^{*} + (1 - \gamma^{*}) Y^{o}$$





EDC reaction rate depends on turbulent flow properties and chemical kinetics approach

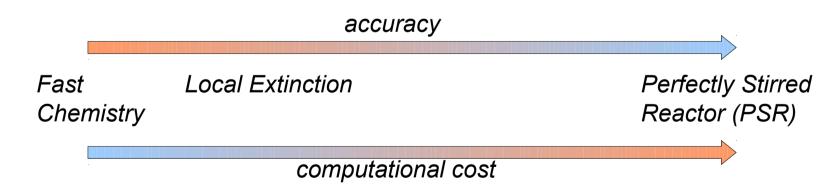
The fraction of the flow occupied by *fine structures:*

$$\gamma^* = 9.7 \left(\frac{\nu \cdot \epsilon}{k^2} \right)^{\frac{3}{4}}$$

Mass transfer rate between the *fine structures* and the *surroundings:*

$$\dot{m}^* = 2.45 \left(\frac{\epsilon}{\nu}\right)^{\frac{1}{2}}$$

Chemical kinetics approaches for fine structure composition Y_i*:



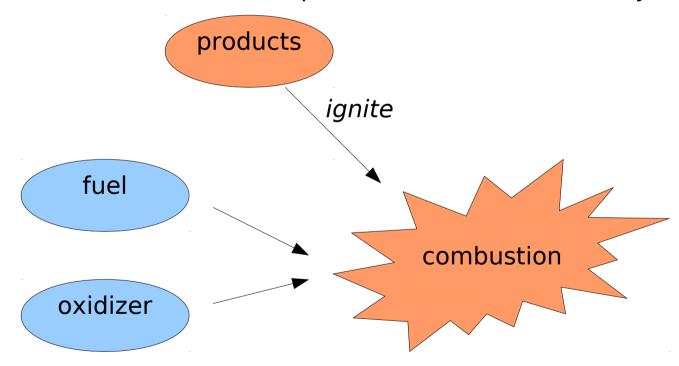




The Fast Chemistry approach assumes infinitely fast reactions

- Assumes sufficient time to achieve equilibrium inside fine structures
- Works only with irreversible global reactions

Combustion occurs if fuel, oxidizer and products meet simultaneously

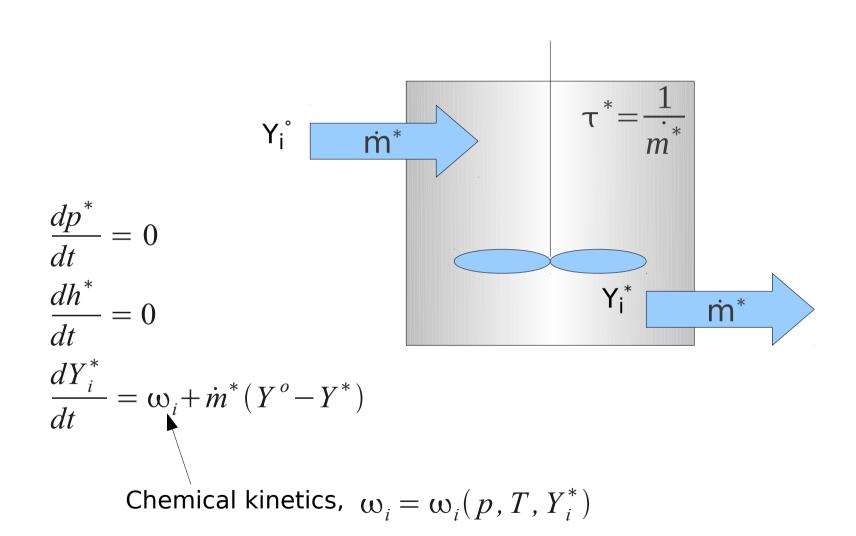


→ Product mass fractions must be initialized accordingly





The PSR approach determines the steady-state of a perfectly stirred reactor







Local Extinction approach employs data from a priori PSR calculations

$$\tau^* < \tau_{ch} \Rightarrow R = 0$$

 τ_{ch} is the minimum residence time which sustains combustion in PSR.

Example 1: Close to burner

high turbulence
$$(\tau^*=2.e-6)$$

$$T = 300 \text{ K}$$

 $(\tau_{ch} = 1.e-4)$

Example 2: Free stream reaction zone

medium turbulence
$$(\tau^*=2.e-4)$$

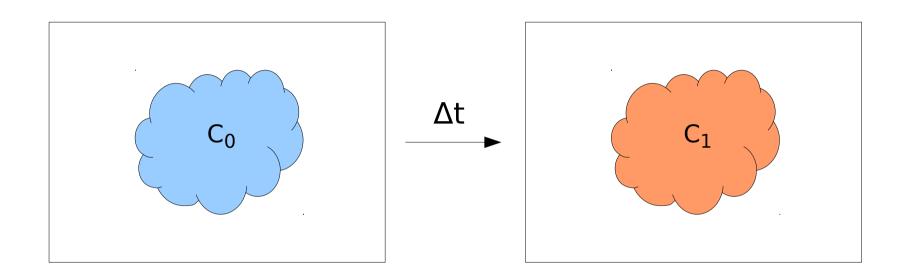
>
$$T = 900 \text{ K}$$

 $(\tau_{ch} = 2.e-5)$





The PaSR combustion model derives the reation rate in a transient manner



$$\bar{R}_i = \kappa \, \frac{C_{i,1} - C_{i,0}}{\Delta \, t}$$

Mixed fraction of cell that can react: κ





The parameter κ is based on two time scales

Turbulent mixing time scale:

Chemical time scale (infinite or finite rate):

$$\tau_m = \sqrt{\frac{k}{\epsilon} \left(\frac{\nu}{\epsilon}\right)^{\frac{1}{2}}}$$

$$\frac{1}{\tau_{ch}} = max \left(-\frac{R_{fuel}}{\rho Y_{fuel}}, -\frac{R_{O_2}}{\rho Y_{O_2}} \right)$$

$$\frac{1}{\tau_{ch}} = -\frac{\partial R}{\rho \, \partial Y}$$





Mixed fraction that reacts:

$$\kappa = \frac{\tau_{ch}}{\tau_m + \tau_{ch}}$$





In OpenFOAM, mixing time scale is implemented slightly different

In rhoReactingFoam:

In Chomiak (1996):

$$\tau_m = C_{mix} \sqrt{\frac{\mu_{eff}}{\rho \epsilon}}$$

$$\tau_m = \sqrt{\frac{k}{\epsilon} \left(\frac{\mathbf{v}}{\epsilon}\right)^{\frac{1}{2}}}$$

Both can be transformed into each other, using:

$$\frac{\mu_t}{\rho} = C_{\mu} \frac{k}{\epsilon}, \quad Sc_t = 1, \quad Re_t = \frac{k^2}{\epsilon \nu}$$

As result, we obtain:

$$C_{mix} = \sqrt{\frac{1}{1 + C_{\mu} R e_t}}$$





The value for C_{mix} needs to be estimated a priori

$$C_{mix} = \sqrt{\frac{1}{1 + C_{\mu} R e_{t}}}$$

Laminar flow

 $Re_t = 0$

 $C_{mix} = 1.0$

Typical turbulent flow

 $Re_t \approx 1000$

 $C_{\text{mix}} \approx 0.1$

Extremely turbulent flow

 $Re_t \rightarrow \infty$

 $C_{mix} \rightarrow 0.0$

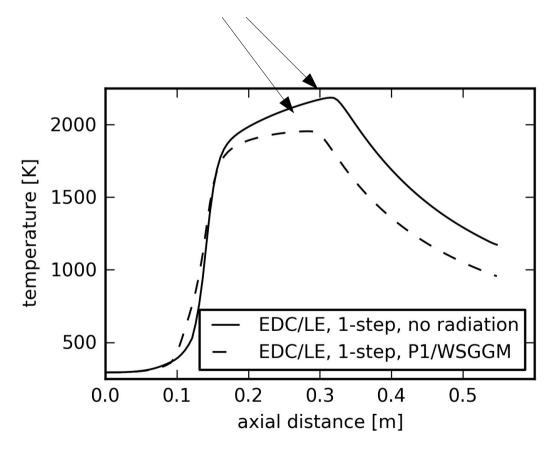
Typical values for C_{mix} : 0.001 – 0.3; cf. Nordin (2001)





Radation heat transfer needs to be considered in combustion simulation

Peak temperature ≈250 K higher without radiation modelling



Radiation Transport Equation:

P1 - Transport

Discrete Ordinates (DOM)

Gas-Absorption Modelling:

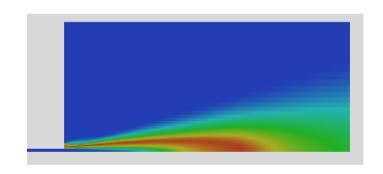
constant

RADCAL-Polynomials

WSGGM (custom)







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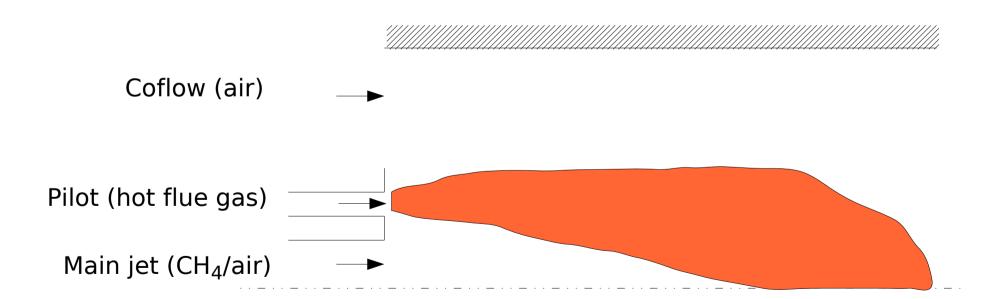
The tutorial case is a non-premixed piloted flame ("Flame D")

Characteristics:

Steady-state, piloted, methane/air, diffusion flame, some local extinction

Geometry:

Axi-symmetric, 2D







The boundary conditions are identical for rhoReactingFoam and rhoSimpleFoam

	Ux [m/s]	p [Pa]	T [K]	Yi [-]	k [m²/s²]	epsilon [m²/s³]
Main jet	49.6	zeroGradient	294	fixedValue	I _{turb} =0.0458	L _{turb} =5.04e-4
Pilot	11.4	zeroGradient	1880	fixedValue	I _{turb} =0.0628	L _{turb} =7.35e-4
Coflow	0.9	zeroGradient	291	fixedValue	l _{turb} =0.0471	L _{turb} =0.0197
Outlet	zeroGradient	100000	zeroGradient	zeroGradient	zeroGradient	zeroGradient
Walls	0.0	zeroGradient	zeroGradient	zeroGradient	wall function	wall function

Front and backside of axi-symmetric domain are specified as 'wedge'.





edcSimpleFoam: Flow field initialized as required by chemical kinetics approach

- Fast Chemistry and Local Extinction: Set CO_2 and H_2O mass fraction to 0.01 everywhere.
- Perfectly Stirred Reactor: Initialize with Fast Chemistry or Local Extinction solution.
- Setup chemistryProperties:

```
edcFastChemCoeffs
{
    oxidiserName 02;
    mainFuelName CH4;
}

no local extinction
above this
temperature

edcLECoeffs
{
    oxidiserName 02;
    mainFuelName CH4;
    autoIgnitionTemperature
    868;

curve1
{
    temperature 300;
    tauChMin 7.00E-005;
}
```

```
edcPSRCoeffs
{
    relativeTolerance 1.e-6;
    absoluteTolerance 1.e-14;
    maxIterations 1.e8;

    useBinaryTree off;
    binaryTreeTolerance 1e-4;
    binaryTreeSize 1.e7;
}
```





rhoReactingFoam: Choosing C_{\min} and ODE intergrator

Estimate turbulent Reynolds number:

$$Re_t$$
=500 $\rightarrow C_{mix}$ = 0.15

Setup chemistryProperties:

```
odeCoeffs
{
    ODESolver SIBS;
    eps 5.0e-4;
    scale 1.0;
}
```

SIBS is stable enough for solving detailed chemistry





Setting-up discretization schemes

```
Convective term: Linear upwind discretization (2<sup>nd</sup> order accurate)
    default    Gauss linearUpwind cellLimited Gauss linear 1;

For species Y<sub>i</sub> (for rhoReactionFoam: Y<sub>i</sub> and hs)
    div(phi, Yi) Gauss multivariateSelection
    {
        //hs        linearUpwind cellLimited Gauss linear 1;
        CH4        linearUpwind cellLimited Gauss linear 1;
        02        linearUpwind cellLimited Gauss linear 1;
        ...
}
```





Setting-up fvSolution

Numerical solver precision depends on solver type:

Transient solver requires each time-step to be accurate all variables: relTol = 0.;

Steady state solver can reach solution through intermediate results

pressure: relTol 0.001;

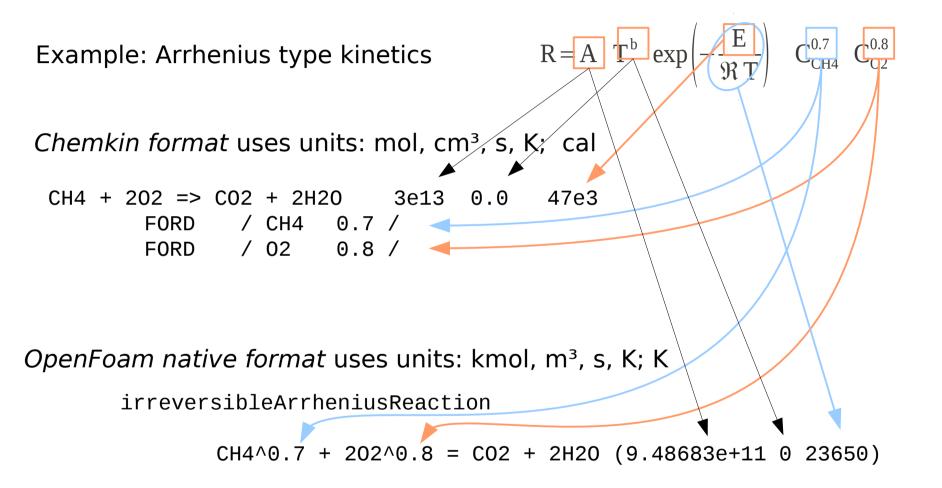
other variables: relTol 0.1;

intermediate results will not be accurate





Chemical mechanisms can be defined in Chemkin or OpenFOAM native format



Use chemkinToFoam to convert chemkin files (or to check their consistency)





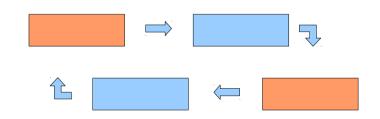
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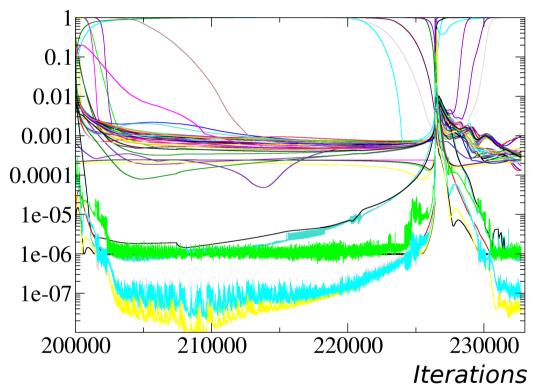






Combustion simulation often faces stability issues

Residuals



Many error sources are possible because numerous models are applied simultaneously, for example:

- Compressible flow
- Coupling of transport equations
- Numerically stiff reaction mechanisms





Solution strategies include good initialization and under-relaxation

Possible "well initialized" flow fields are:

- Cold flow
- Starting solution (steady-state; 1- or 2-Step)
- Products

Strong coupling between transport equations may be broken with different under-relaxation factors.

species: 0.2 (reaction model)



temperature: 0.7





pressure/ velocity: 0.3/0.7

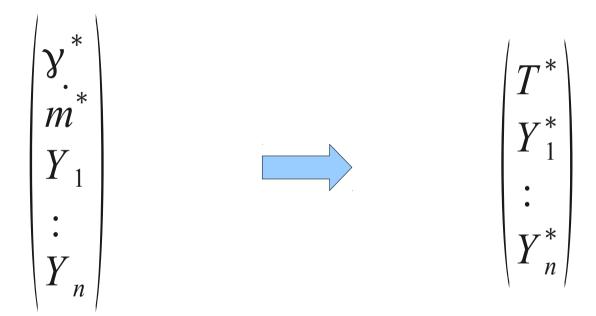


density: 0.1

Under-relaxation only applicable to steady-state cases. Unsteady solver based on Transient-SIMPLER needed?





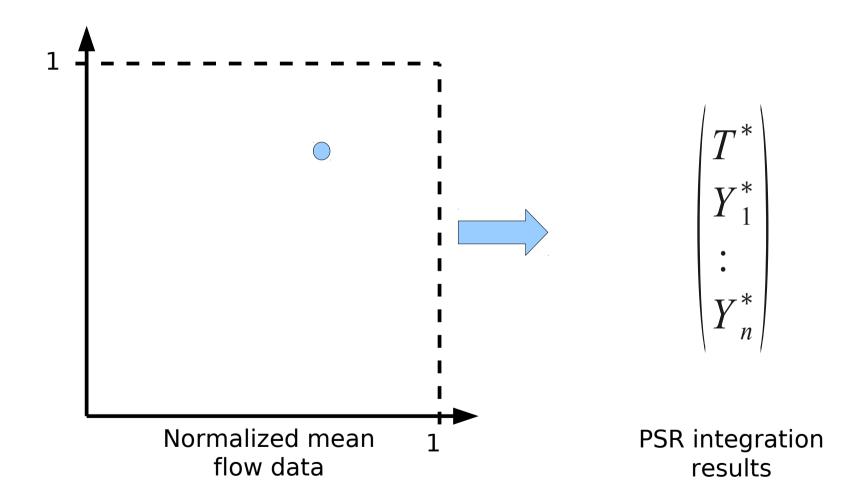


Mean flow data

PSR integration results

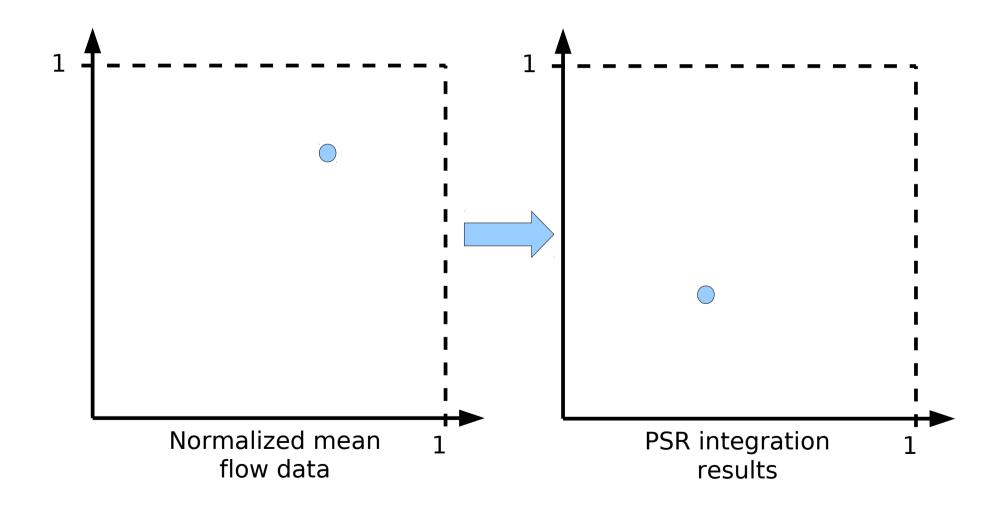






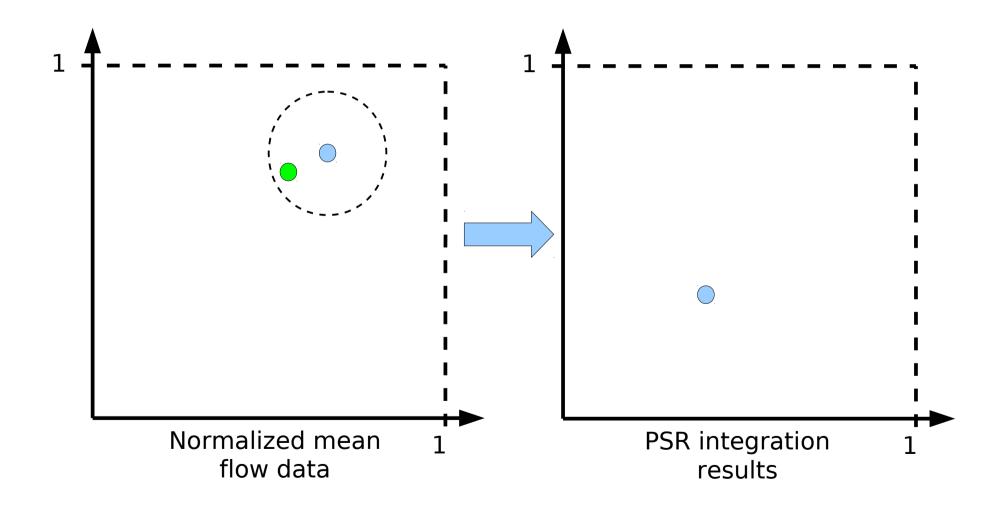






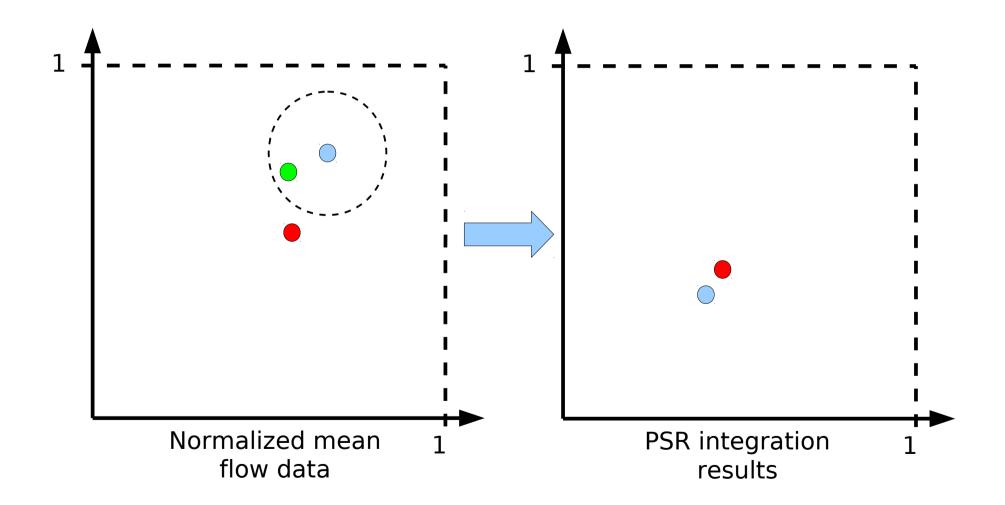






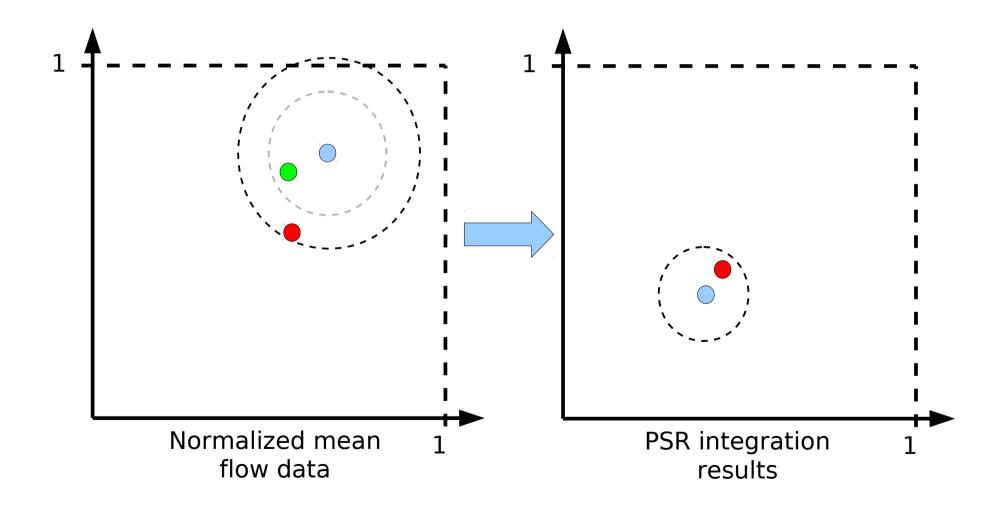
















Limiting temperature is possible in steady-state cases

In a steady state case, intermediate "time steps" are not accurate.

→ Temperature may temporarily increase and needs to be limited

Solver level implementation in edcSimpleFoam:

- New enthalpy field calculated from species mass fractions and T_{min}
- Another enthalpy field calculated for T_{max}
- Both fields are used to limit enthalpy field
- T_{\max} and T_{\min} are specified in thermophysical Properties

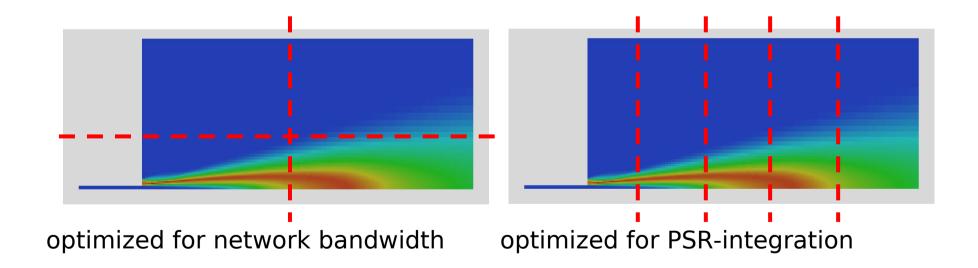
Request for integrated limitation filed in OpenCFD's bugtracker (Issue #57).





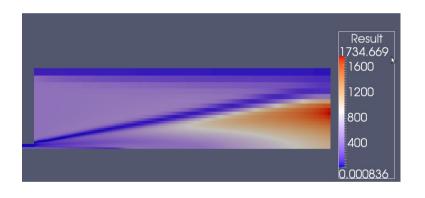
Optimal parallelization depends on the complexity of the chemical model

Integrating complex chemical mechanisms is computationally much more expensive than solving transport equations (even if there are many)!









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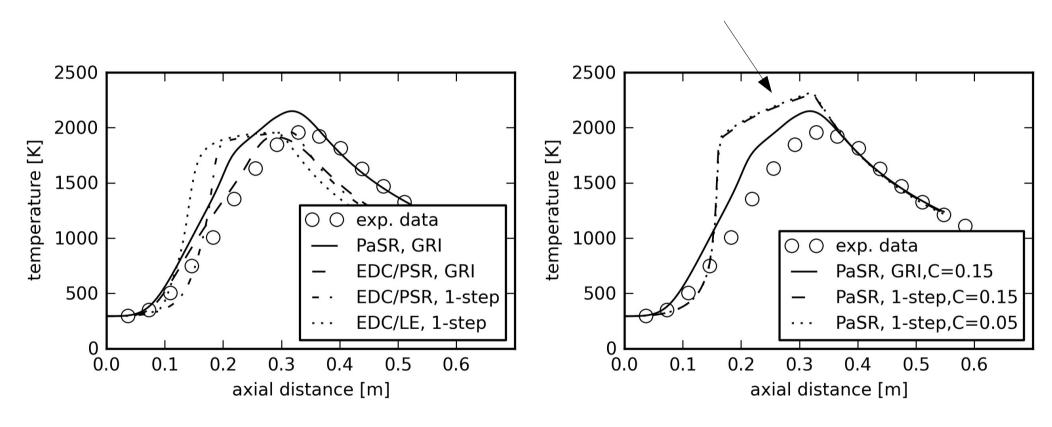
Validation





Detailed reaction mechnism predicts temperature profile accurately





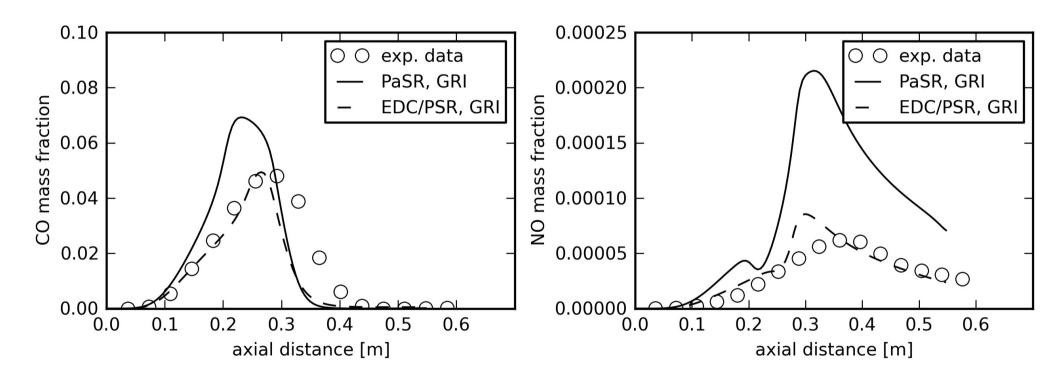
Radiation modeling used with EDC, not used with PaSR.

exp. data: Barlow, R. S. and Frank, J. H., Proc. Combust. Inst. 27:1087-1095 (1998)





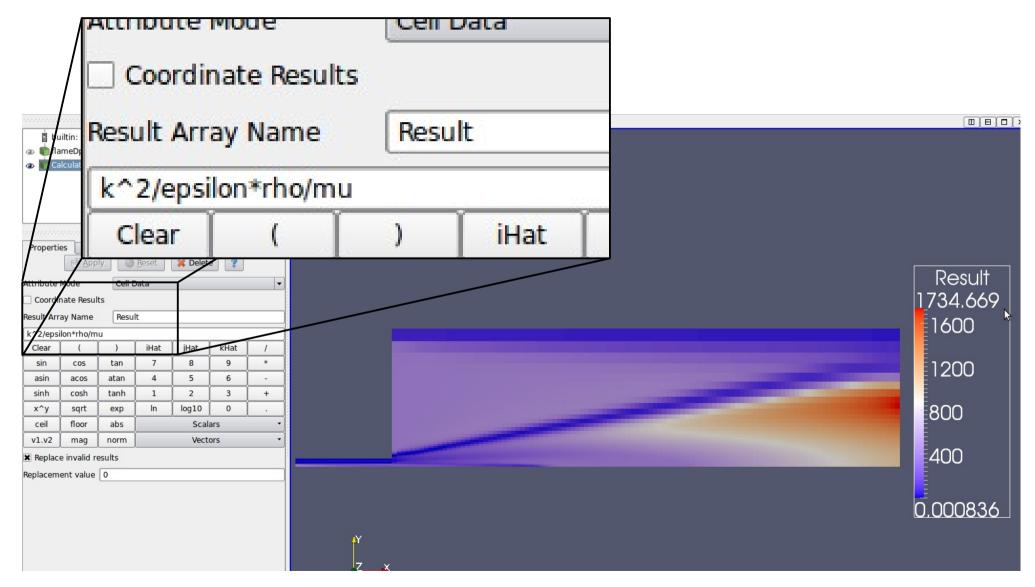
Intermediate species and pollutants are more difficult to predict







ParaFoam's "calculator" can be used to check Ret assumption







Comparison with measurements may require special postprocessing

Common difficulties when comparing simulated mass fractions with measurements:

- Measured data are often mole fractions or concentrations If not all (major) species are measured, correct conversion to mass fractions impossible
- Flue gas or emission monitoring can be measured in "dry gas", i.e. after water vapor has been condensed out Simulated data comprise a complete set, therefore they can be accurately converted

New utility massToMoleFraction handles conversion together with "-dryGas" option





Final note: When using edcSimpleFoam or edcPisoFoam, please cite

for the EDC model:

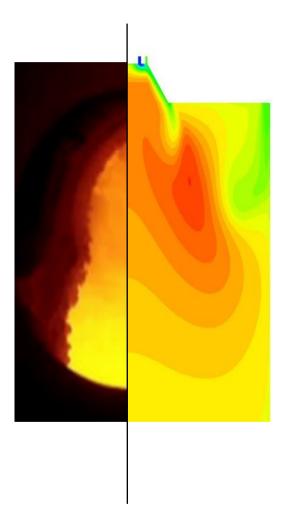
B. Magnussen: The Eddy Dissipation Concept: A Bridge between Science and Technology, ECCOMAS Thematic Conference on Computational Combustion, Lisbon, Portugal, 2005

for the validation of the OpenFoam implementation:

B. Lilleberg, D. Christ, I.S. Ertesvåg, K.E. Rian, R. Kneer, Numerical simulation with an extinction database for use with the Eddy Dissipation Concept for turbulent combustion (submitted)







Thank you for your attention!



